

Dynamics of Systems

CTB 2300

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1. Free vibration of an SDOF with viscous damping:

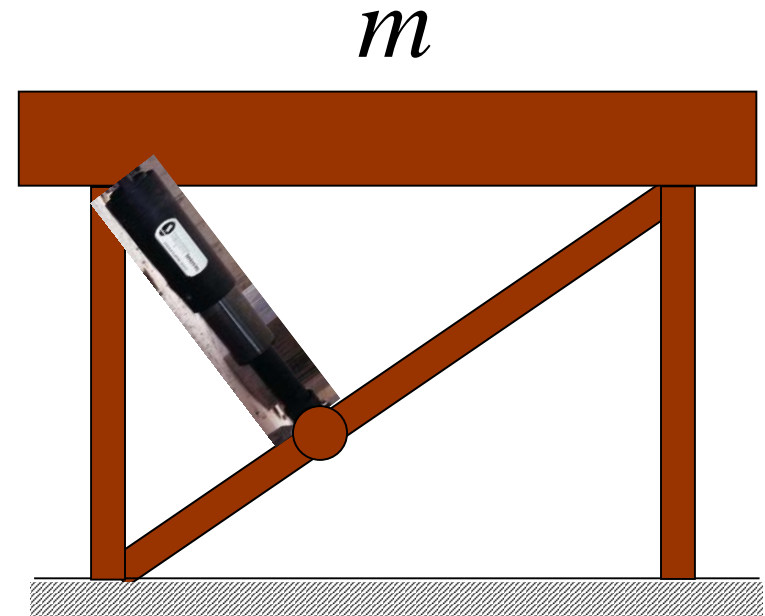
- Viscous dampers: impression
- Derivation of the equation of motion
- The characteristic equation and motion types
- Super-critically damped motion
- Critically damped motion
- Sub-critically damped vibration
- Measurement of the damping ratio

Viscous dampers: impression



Viscous damper in a building

Viscous dampers: impression



Viscous damper in a frame structure

Video: [Dampers for earthquake protection](#)

Viscous dampers: impression



Seismic fluid viscous dampers for large highway bridge, 1.5 million pounds output force (how large they can be!)

Viscous dampers: impression



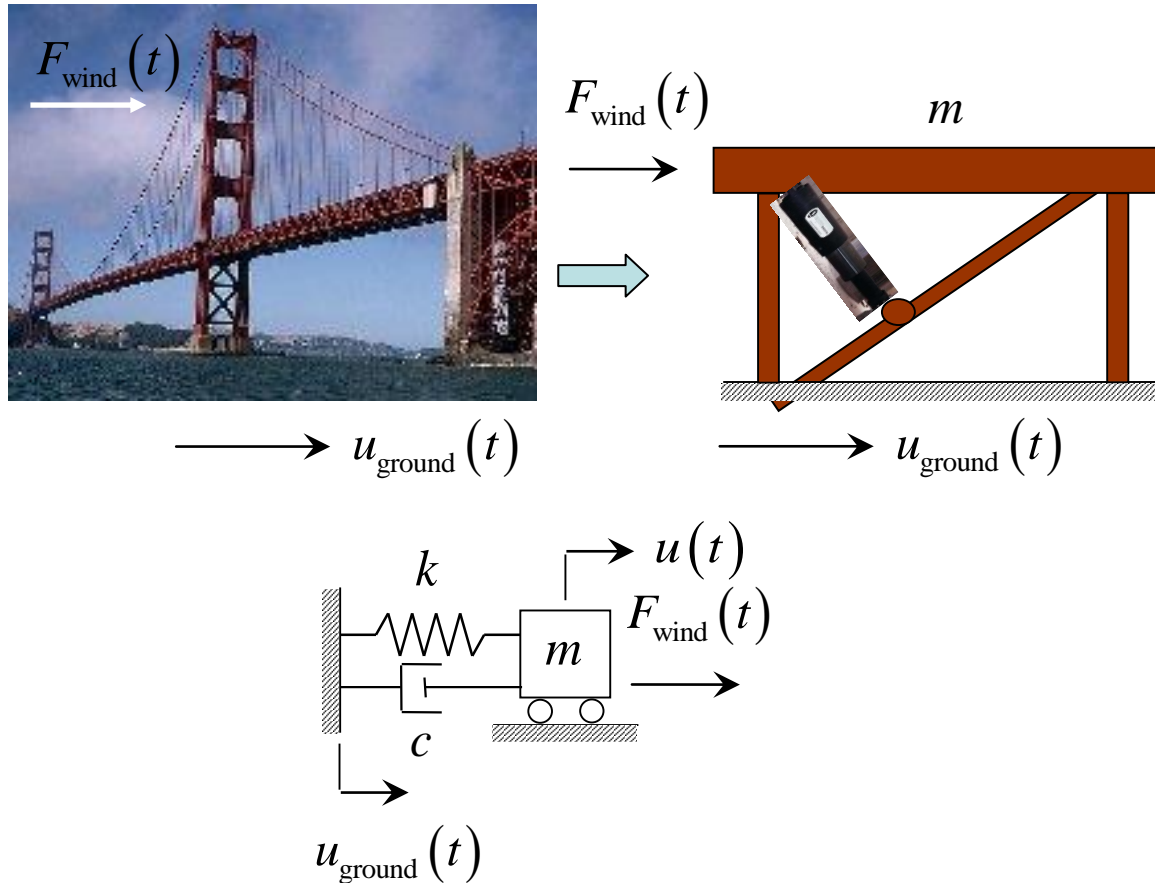
Viscous dampers at a cable anchorage for a cable-stayed bridge

Viscous dampers: impression

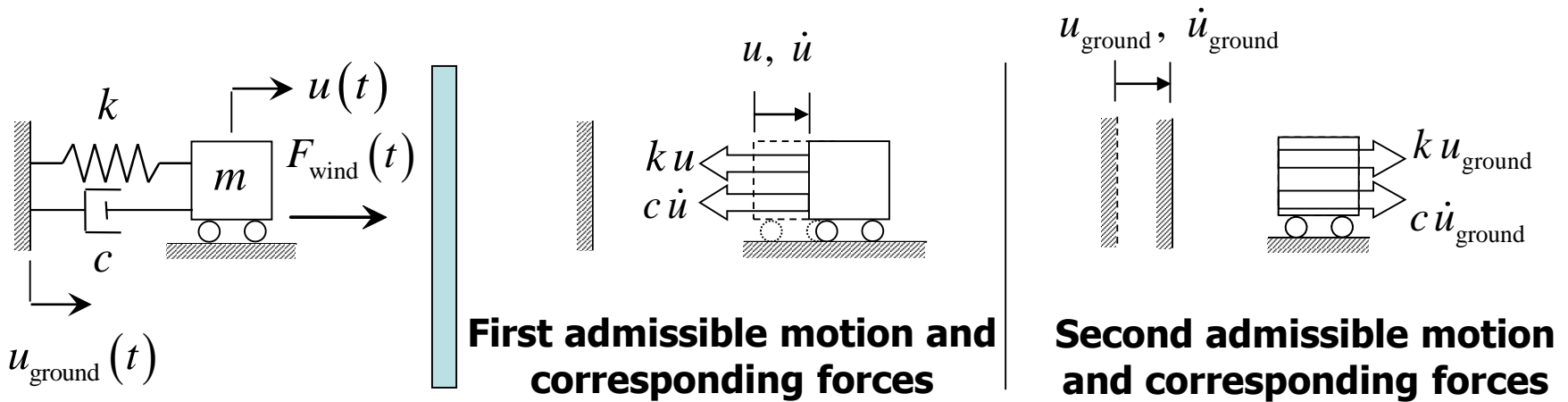


Viscous dampers for the Millennium bridge in London

SDOF with viscous damping



Derivation of the equation of motion



Resulting equation of motion:

$$m\ddot{u} = -k u + k u_{\text{ground}} - c \dot{u} + c \dot{u}_{\text{ground}} + F_{\text{wind}}(t)$$



$$m \ddot{u} + c \dot{u} + k u = F(t),$$

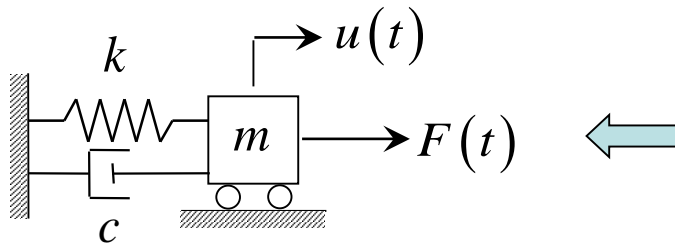
$$F(t) = k u_{\text{ground}} + c \dot{u}_{\text{ground}} + F_{\text{wind}}(t)$$

mass \times acceleration = force

Damped SDOF: Generalized equation of motion and the set of governing equations

The governing equations for damped SDOF systems: $m \ddot{u} + c \dot{u} + k u = F(t)$ ← Equation of motion

$$\left. \begin{aligned} u(0) &= u_0 \\ \dot{u}(0) &= v_0 \end{aligned} \right\} \leftarrow \text{Initial conditions}$$



← Schematization of a generalized SDOF with viscous damping (the damped SDOF)

Free vibration of the damped SDOF: equation of motion

Equation of motion:

$$m \ddot{u} + c \dot{u} + k u = 0$$

Canonical form of equation of motion:

$$\ddot{u} + 2\zeta \omega_0 \dot{u} + \omega_0^2 u = 0,$$

$\omega_0 = \sqrt{k/m}$ is the natural frequency

$\zeta = c/c_{\text{critical}}$ is the damping ratio

$c_{\text{critical}} = 2\sqrt{k m}$ is the critical damping

The characteristic equation and possible motion types

Equation of motion:

$$\ddot{u} + 2\zeta \omega_0 \dot{u} + \omega_0^2 u = 0,$$

The general solution:

$$u(t) = \sum_{n=1}^2 U_n \exp(s_n t)$$

Characteristic equation and its roots:

$$s_n^2 + 2\zeta \omega_0 s_n + \omega_0^2 = 0, \quad s_{1,2} = \omega_0 \left(-\zeta \pm \sqrt{\zeta^2 - 1} \right)$$

Possible types of motion:

aperiodic motion:	⎧ $\zeta > 1$ (super-critically damped system, I) $\zeta = 1$ (critically damped system, II)
damped vibration:	

I. The super-critically damped system (aperiodic motion)

The roots of the characteristic equation:

$$s_{1,2} = \omega_0 \left(-\zeta \pm \sqrt{\zeta^2 - 1} \right) \text{ are real and negative}$$

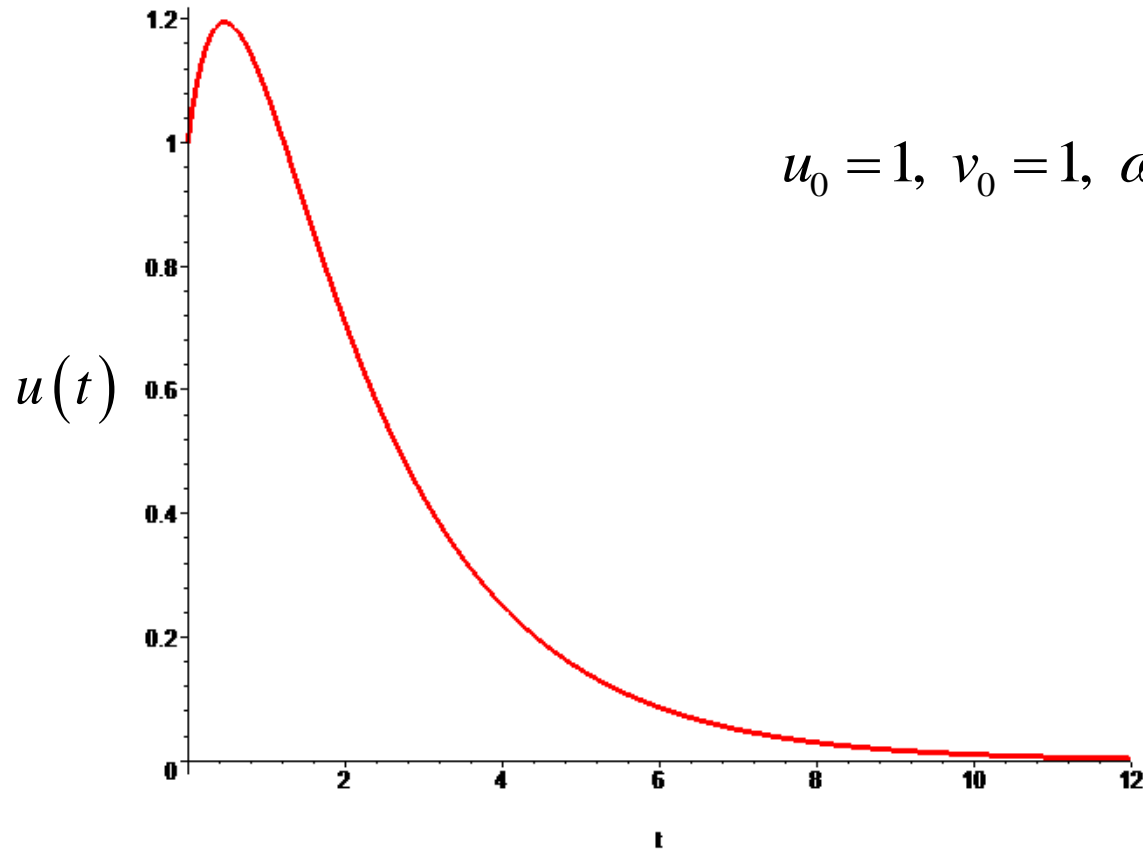
The general solution to the equation of motion:

$$u(t) = \exp(-\zeta\omega_0 t) \left(A \exp\left(\omega_0 t \sqrt{\zeta^2 - 1}\right) + B \exp\left(-\omega_0 t \sqrt{\zeta^2 - 1}\right) \right)$$

The solution to the governing equations (with the initial conditions):

$$u(t) = \frac{\exp(-\zeta\omega_0 t)}{2\sqrt{\zeta^2 - 1}} \left\{ \left(u_0 \omega_0 \left(\sqrt{\zeta^2 - 1} + \zeta \right) + v_0 \right) \exp\left(\omega_0 t \sqrt{\zeta^2 - 1}\right) + \left(u_0 \omega_0 \left(\sqrt{\zeta^2 - 1} - \zeta \right) - v_0 \right) \exp\left(-\omega_0 t \sqrt{\zeta^2 - 1}\right) \right\}$$

I. The super-critically damped system (aperiodic motion)



$$u_0 = 1, v_0 = 1, \omega_0 = 1, \zeta = 1.2$$

To understand the effect of the parameters on the motion, please use the MAPLE file to this lecture

II. The critically damped system (aperiodic motion)

The roots of the characteristic equation:

$s_{1,2} = -\omega_0$ are the same (multiple); real and negative

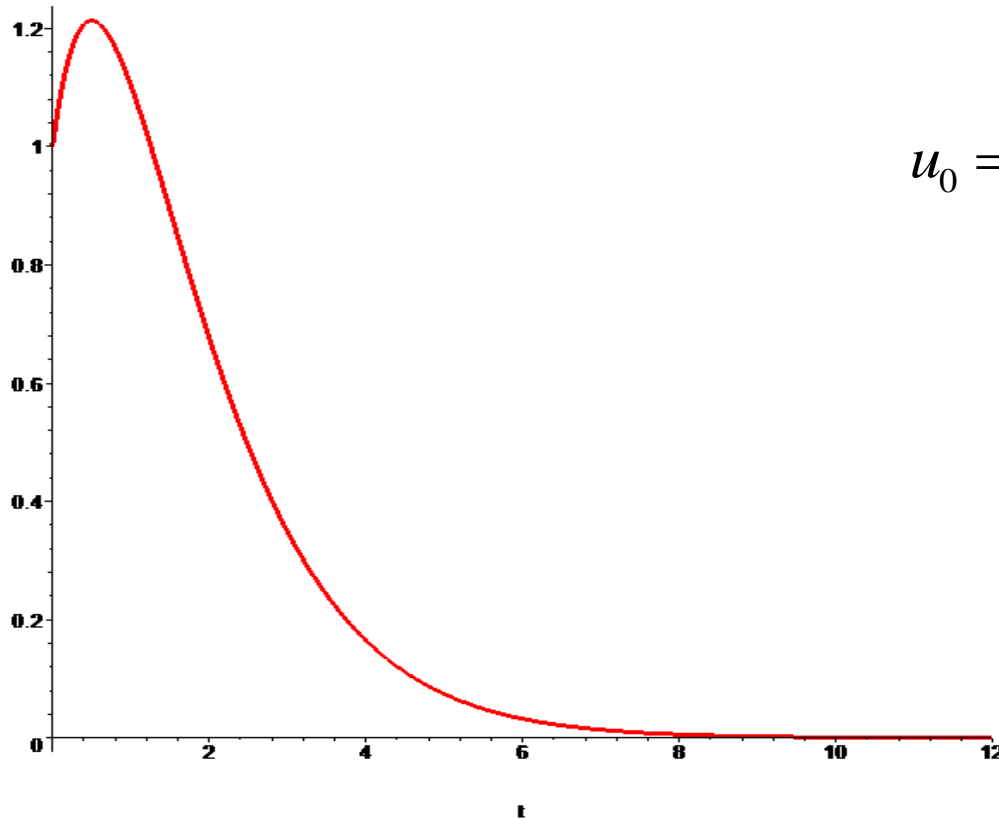
The general solution to the equation of motion:

$$u(t) = \exp(-\omega_0 t)(A + Bt)$$

The solution to the governing equations (with the initial conditions):

$$u(t) = \exp(-\omega_0 t) \{ u_0 + (v_0 + \omega_0 u_0) t \}$$

II. The critically damped system (aperiodic motion)



$$u_0 = 1, v_0 = 1, \omega_0 = 1$$

To understand the effect of the parameters on the motion, please use the MAPLE file to this lecture

III. The sub-critically damped system (damped vibration)

The roots of the characteristic equation:

$$s_{1,2} = \omega_0 \left(-\zeta \pm i\sqrt{1-\zeta^2} \right) \text{ are complex-conjugate (real part is negative)}$$

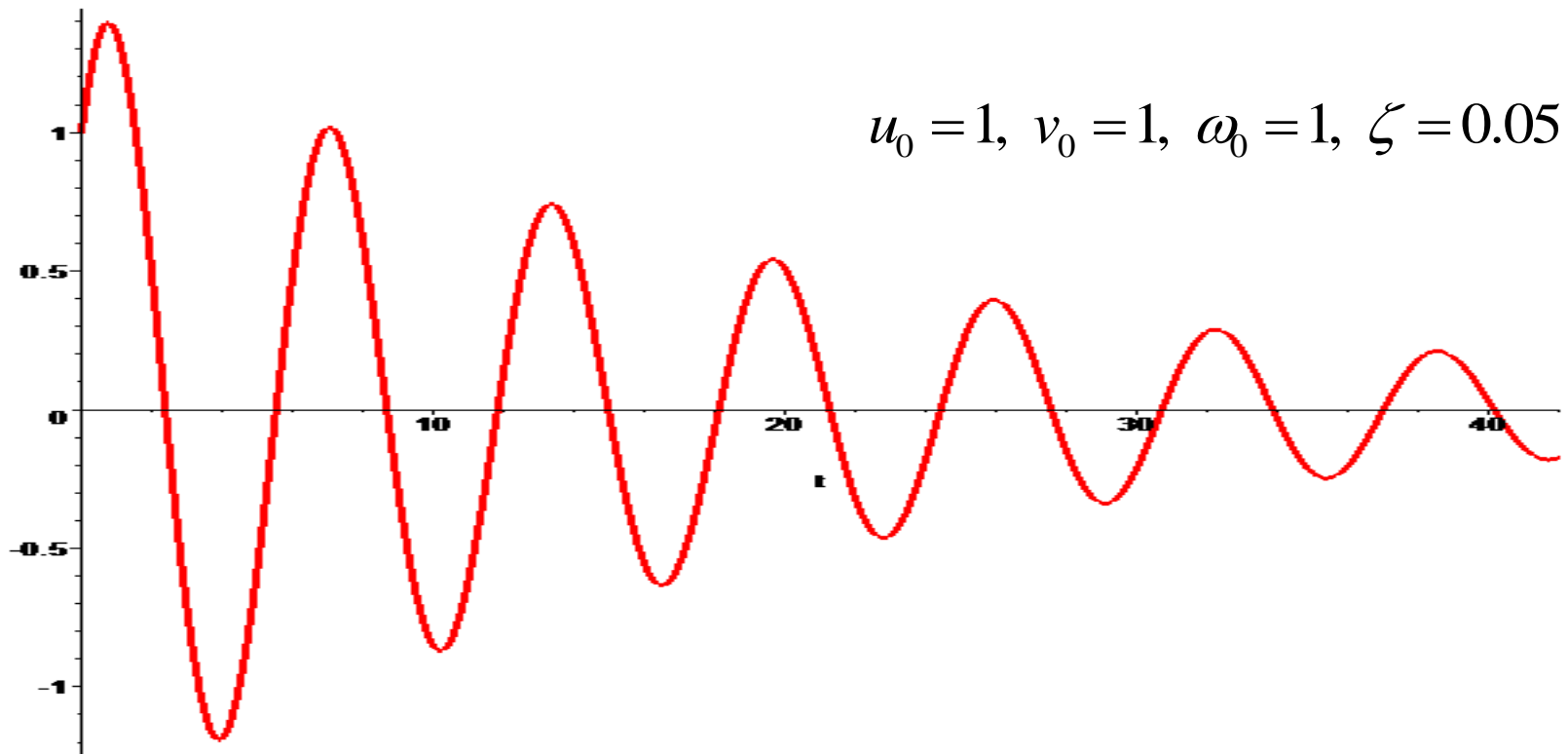
The general solution to the equation of motion:

$$u(t) = \exp(-\zeta\omega_0 t) \left(A \cos(\omega_1 t) + B \sin(\omega_1 t) \right), \quad \omega_1 = \omega_0 \sqrt{1-\zeta^2}$$

The solution to the governing equations (with the initial conditions):

$$u(t) = \exp(-\zeta\omega_0 t) \left(u_0 \cos(\omega_1 t) + \frac{v_0 + \zeta\omega_0 u_0}{\omega_1} \sin(\omega_1 t) \right)$$

III. The sub-critically damped system (damped vibration)



To understand the effect of the parameters on the motion, please use the MAPLE file to this lecture

III. Damped vibration: a compact form of the solution

The solution:

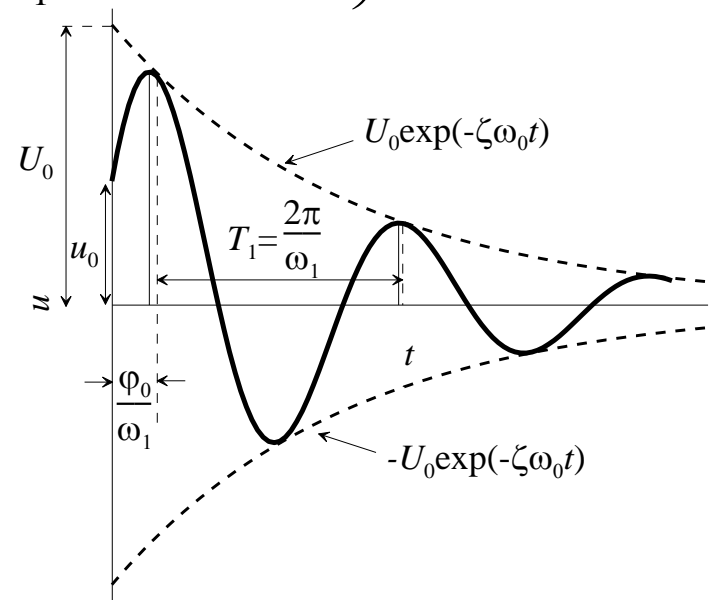
$$u(t) = \exp(-\zeta\omega_0 t) \left(u_0 \cos(\omega_1 t) + \frac{v_0 + \zeta\omega_0 u_0}{\omega_1} \sin(\omega_1 t) \right)$$

Alternative form of the solution:

$$u(t) = U_0 \exp(-\zeta\omega_0 t) \cos(\omega_1 t + \varphi_0)$$

$$U_0 = \sqrt{u_0^2 + \left(\frac{v_0 + \zeta\omega_0 u_0}{\omega_1} \right)^2}$$

$$\varphi_0 = -\arctan \left(\frac{v_0 + \zeta\omega_0 u_0}{u_0 \omega_1} \right)$$



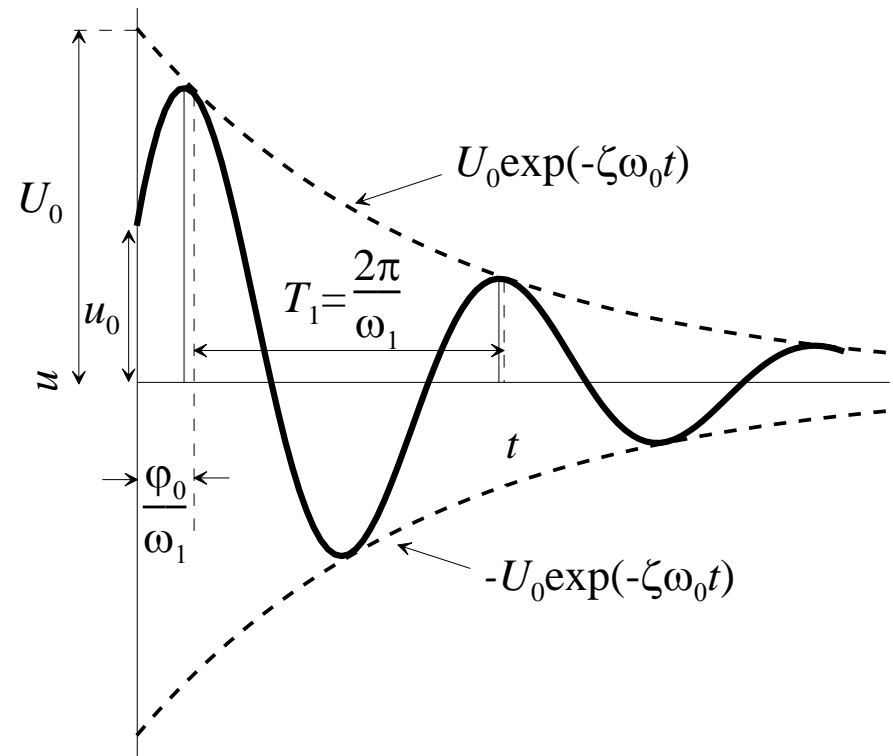
III. Damped vibration: the amplitude, damped frequency, period and phase angle

$$u(t) = U_0 \exp(-\zeta\omega_0 t) \cos(\omega_1 t + \varphi_0)$$

Amplitude: $U_0 \exp(-\zeta\omega_0 t)$

Period: $T_1 = 2\pi/\omega_1 = \frac{2\pi}{\omega_0 \sqrt{1-\zeta^2}}$

Phase angle: φ_0



III. Damped vibration: how can we measure the damping ratio?

Measurement of damping.

Let u_1 be a displacement at t
and u_s a displacement after $(s-1)$ cycles.

Then

$$\frac{u_1}{u_s} = \exp((s-1)\zeta\omega_0 T_1)$$

or $\log\left(\frac{u_1}{u_s}\right) = (s-1)\zeta\omega_0 T_1$

$$\zeta\omega_0 T_1 = \frac{2\pi\zeta}{\sqrt{1-\zeta^2}} - \text{logarithmic decrement}$$

$$u(t) = U_0 \exp(-\zeta\omega_0 t) \cos(\omega_1 t + \varphi_0)$$

The frequency of damped vibrations is normally approximated by the natural frequency of the undamped system

The reason:

$$\omega_1 = \omega_0 \sqrt{1-\zeta^2} \approx \omega_0 \left(1 - \frac{1}{2}\zeta^2\right) \approx \omega_0$$

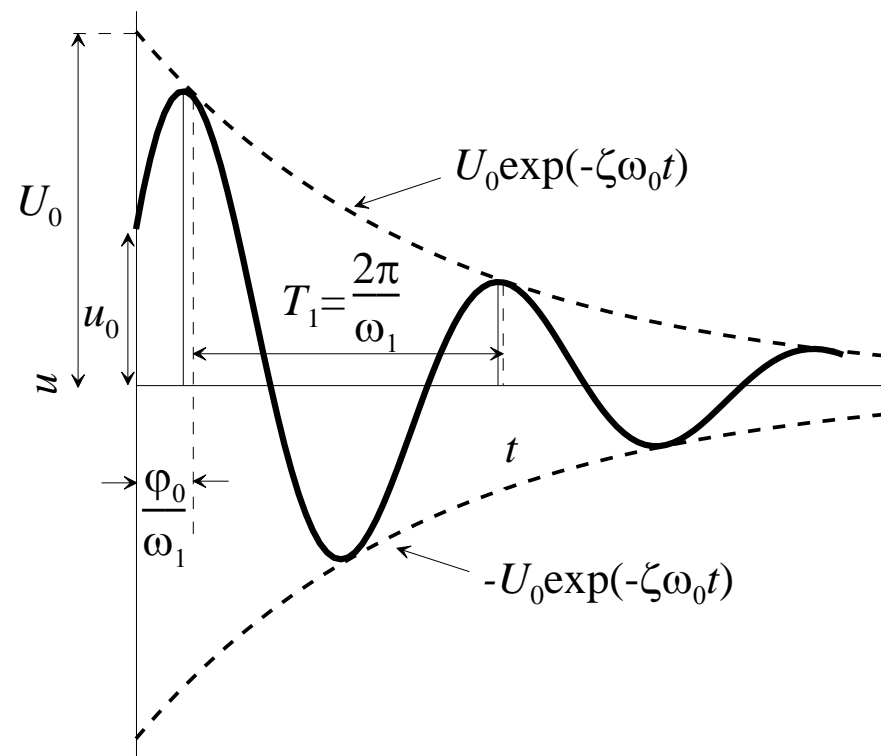
Exercise: compute the damping ratio from observation

We consider the free vibration of a sub-critically damped SDOF system, as shown in the figure.

Suppose that the amplitude decay over 1 natural period is equal to α .

Q1: derive the damping ratio ζ in terms of α .

Q2: is the result influenced by the specific initial conditions that excited the motion?



$$u(t) = U_0 \exp(-\zeta\omega_0 t) \cos(\omega_1 t + \varphi_0)$$